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EVALUATION OF TURBULENCE MODELS IN MHD CHANNEL AND SQUARE DUCT FLOWS

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ABSTRACT

In this paper, several low and high Reynolds number versions of k - ϵ and Reynolds stress turbulence models have been evaluated in a channel and a square duct flow with and without a magnetic field by comparing the predictions with direct numerical simulations data. The simulations are performed using FLUENT solver. The additional source terms for magnetic field effects on turbulence have been included through user-defined functions. A systematic assessment of the predicted mean flow, turbulence quantities, frictional losses and computational costs of the various turbulence models is presented.

All the models predict mean axial velocity reasonably well, but the predictions of turbulence parameters are less accurate. Velocity predictions are worse for the square duct flow due to secondary flows generated by the turbulence. The implementation of the MHD sources generally improves predictions in MHD flows, especially for low-Re k - ϵ models. The high-Re models using the wall treatments show little improvement, perhaps due to the lack of MHD effects in the wall formulations. Finally, at low Reynolds numbers, the Lam-Bremhorst (LB) low-Re k - ϵ model was found to give better predictions than other models for both hydrodynamic and magnetic field influenced turbulent flows.

NOMENCLATURE

$R_{ij} = -\overline{u'_i u'_j}$ Reynolds stresses (m^2/s^2)
 $\overline{F_L}$ Average Lorentz force (N/m^3)
 μ Dynamic viscosity ($\text{N}\cdot\text{s}/\text{m}^2$)
 μ_t Turbulent dynamic viscosity ($\text{N}\cdot\text{s}/\text{m}^2$)
 $\nu = \mu/\rho$ Kinematic viscosity (m^2/s)
 ν_t Turbulent kinematic viscosity (m^2/s)

ρ Density (kg/m^3)
 t Time (s)
 x Spatial coordinate (m)
 \overline{u} Ensemble average velocity (m/s)
 u' Velocity fluctuations (m/s)
 \overline{p} Ensemble average pressure (N/m^2)
 p' Pressure fluctuations (N/m^2)
 k Turbulent kinetic energy (m^2/s^2)
 ϵ Turbulent dissipation rate (m^2/s^3)
 G_k Turbulent kinetic energy production rate ($\text{kg}/(\text{m}\cdot\text{s}^3)$)
 y Normal distance from wall (m)
 $y^+ = yu_\tau/\nu$ Normalized wall normal distance
 $u_\tau = \sqrt{\tau_w/\rho}$ Friction velocity (m/s)
 τ_w Wall stress (N/m^2)
 Re_m Magnetic Reynolds number
 L Characteristics length (m)
 σ Electrical conductivity ($1/(\text{ohm}\cdot\text{m})$)
 μ_0 Permeability of free space (h/m)
 \vec{v} Velocity vector (m/s)
 ϕ Electric potential (V/m^2)
 ϕ' Electric potential fluctuations (V/m^2)
 $\vec{B}_0 = (B_{x0}, B_{y0}, B_{z0})$ Applied magnetic field vector (Tesla)
 \vec{b} Induced magnetic field vector (Tesla)
 $\vec{B} = \vec{B}_0 + \vec{b}$ Total magnetic field vector (Tesla)
 \vec{J} Current density vector (A/m^2)

$S_{k,\varepsilon,w'w',u'u',v'v',u'v',w'u',w'v'}$	Source terms to k , ε , $\overline{w'w'}$, $\overline{u'u'}$, $\overline{v'v'}$, $\overline{w'u'}$, $\overline{w'v'}$, and $\overline{u'v'}$ equations (kg/m-s^3), where $S_k^{M+} = S_k^M / (\rho u_\tau^2)$.
Re_τ	Friction Reynolds number
Ha	Hartmann number
Re	Bulk Reynolds number
$\frac{\partial \bar{p}}{\partial z}$	Mean axial pressure gradient (N/m^3)
W_b	Bulk axial velocity (m/s)
ε_{kmn}	Levi-Civita symbol
δ_{ij}	Kronecker's delta
$\frac{\partial}{\partial n}$	Partial derivative in wall normal direction ($1/\text{m}$)
i, j, k	Index notation
x, y, z	Spatial coordinates (m)

INTRODUCTION

Reynolds-Averaged Navier-Stokes (RANS) simulations are widely used to optimize various industrial flows because of their low computational cost. However, it is well-known that their accuracy in complex flows is limited by the difficulties in modeling the complex turbulence interactions through transport equations for the mean flow variables [1]. Significant effort has already been devoted to validation, improvement, and custom tailoring of these models of turbulent flows for different classes of flows [2-8]. This is usually done through comparisons with experimental data. However, with the availability of Direct Numerical Simulation (DNS) and Large Eddy Simulation (LES) computed flow fields, it has also become possible to evaluate the turbulence models using DNS / LES data [2, 9-11].

Despite the importance of magnetic fields in material processing, very limited work [12-15] exists on improving and testing turbulence models to include the effects of a magnetic field on the turbulence. A few modified models with magnetic field effects have been tested in channel flow/rectangular duct flow with a partial magnetic field (low-Re k- ε and Reynolds stress model (RSM)) [12-13], pipe flow (low-Re k- ε) [14] and free surface channel flow (k- ε) [15]. The modifications proposed in the latter two of these studies (pipe flow [14] and free surface channel flow [15]) were based upon bulk properties of the flow and cannot be generalized to other flows. The first two studies (k- ε and RSM, [12-13]) relate the magnetic field generated source terms in the turbulent transport equations to the local properties, and therefore can be generalized to other flows. However, these models have been so far tested only in a turbulent channel flow and in a rectangular duct with a partial magnetic field. For the rectangular duct with a partial magnetic field only the mean velocity was compared. The mean velocity obtained with this model was reported to show better agreement with measurements but no comparisons are available for turbulence quantities [12].

The present work reports a systematic assessment of a number of turbulence models, and their variants, for magnetohydrodynamic (MHD) flow in two representative geometries: a) channel flow, and b) a square duct flow. Confined internal flows through long pipes and ducts are relevant in many commercial flows. The square duct flow is more complicated to predict because of the turbulence-driven secondary flows [16]. The various models considered are: a) 3 variants of high-Re two-equation models (Standard k- ε (SKE) [17], RNG k- ε (RNG) [18], Realizable k- ε (RKE) [19], b) 6

low-Re k- ε models (Abid [20], Lam-Bremhorst (LB) [21], Launder-Sharma (LS) [22], Yang-Shih (YS) [23], Abe-Kondoh-Nagano (AKN) [24], and Chang-Hsieh-Chen (CHC) [25-26]) and c) 2 second-momentum closure Reynolds Stress Models with Linear Pressure Strain (RSM-LPS) and Stress-Omega (RSM-S ω) [27-31]) models along with standard wall functions (SWF) [32], non-equilibrium wall functions (NEWF) [33], and two-layer wall treatment combined with single-blended wall function (enhanced wall treatment (EWT)) [34-35, 30]. The simulations have been performed using FLUENT [30] and the effect of magnetic field on turbulence, as given by Kenjereš and Hanjalić [12-13], has been incorporated through additional source terms using user-defined functions (UDF). Mean velocities, turbulent kinetic energy (TKE), root mean square (RMS) of velocity fluctuations, MHD sources/sinks and frictional losses are compared against available DNS data in channel and square duct flows.

TURBULENCE MODELS TESTED

The ensemble averaged Navier-Stokes equations are solved [36-37]:

$$\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial \bar{u}_i \bar{u}_j}{\partial x_j} = -\frac{1}{\rho} \frac{\partial \bar{p}}{\partial x_i} + \frac{\partial}{\partial x_j} \left(\nu \frac{\partial \bar{u}_i}{\partial x_j} \right) + \frac{\partial R_{ij}}{\partial x_j} + \bar{F}_L \quad (1)$$

where, $R_{ij} = -\overline{u'_i u'_j}$: Reynolds Stresses, and \bar{F}_L is the average Lorentz force due to magnetic field.

k- ε Models

The k- ε models use Boussinesq hypothesis for Reynolds stresses:

$$R_{ij} = -\overline{u'_i u'_j} = \nu_t \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right) - \frac{2}{3} k \delta_{ij}$$

where Kronecker delta, $\delta_{ij} = 1$, if $i=j$, else 0

The base equations for two equation k- ε models are given as;

$$\frac{\partial \rho k}{\partial t} + \frac{\partial}{\partial x_k} (\rho \bar{u}_k k) = \frac{\partial}{\partial x_j} \left[(A) \frac{\partial k}{\partial x_j} \right] + G_k - \rho \varepsilon + \rho D \quad (2)$$

$$\frac{\partial \rho \varepsilon}{\partial t} + \frac{\partial}{\partial x_k} (\rho \bar{u}_k \varepsilon) = \frac{\partial}{\partial x_j} \left[(B) \frac{\partial \varepsilon}{\partial x_j} \right] + C + \rho E \quad (3)$$

Where, $k = \overline{u'_i u'_i} / 2$, $\varepsilon = \nu \frac{\partial \bar{u}'_i}{\partial x_j} \frac{\partial \bar{u}'_j}{\partial x_i}$, and $G_k = -\rho \overline{u'_i u'_j} \frac{\partial \bar{u}_j}{\partial x_i}$

The formulations for A , B , C , D and E along with other parameters for various k- ε turbulence models (standard [17], RNG [18], Realizable [19], low-Re: Abid [20], Lam-Bremhorst (LB) [21], Launder-Sharma (LS) [22], Yang-Shih (YS) [23], Abe-Kondoh-Nagano (AKN) [24], and Chang-Hsieh-Chen (CHC) [25-26]) tested are given elsewhere [17-26].

Reynolds stress models (RSM)

The exact transport equation for the six independent Reynolds stresses ($\overline{u'_i u'_j}$) in RSM can be written as [27-31]:

$$\frac{\partial}{\partial t} (\rho \overline{u'_i u'_j}) + \frac{\partial}{\partial x_k} (\rho \bar{u}_k \overline{u'_i u'_j}) = P_{ij} + D_{ij}^L + D_{ij}^T + \phi_{ij} - \varepsilon_{ij} \quad (4)$$

$$P_{ij} = -\rho \left(\overline{u'_i u'_j} \frac{\partial \bar{u}_j}{\partial x_k} + \overline{u'_j u'_i} \frac{\partial \bar{u}_i}{\partial x_k} \right) \text{ (I: Production),}$$

$$D_{ij}^L = \frac{\partial}{\partial x_k} \left(\mu \frac{\partial}{\partial x_k} (\overline{u'_i u'_j}) \right) \text{ (II: Molecular diffusion),}$$

$$D_{ij}^T = -\frac{\partial}{\partial x_k} \left(\overline{\rho u'_i u'_j u'_k} + \overline{p' (\delta_{kj} u'_i + \delta_{ik} u'_j)} \right) \text{ (III: Turbulent}$$

$$\text{diffusion), } \phi_{ij} = p' \left(\frac{\partial u'_i}{\partial x_j} + \frac{\partial u'_j}{\partial x_i} \right) \text{ (IV: Pressure strain),}$$

$$\varepsilon_{ij} = 2\mu \frac{\partial u'_i}{\partial x_k} \frac{\partial u'_j}{\partial x_k} \text{ (V: Dissipation).}$$

Of these five terms, the last three (D_{ij}^T , ϕ_{ij} and ε_{ij}) require modeling, with the pressure strain (ϕ_{ij}) and dissipation (ε_{ij}) considered to be critical [28]. The turbulent diffusion term (i.e. D_{ij}^T , III) is modeled the same way as the molecular diffusion term [38]:

$$D_{ij}^T = \frac{\partial}{\partial x_k} \left(\frac{\mu_t}{\sigma_k} \frac{\partial}{\partial x_k} (\overline{u'_i u'_j}) \right), \text{ where } \mu_t = \rho C_\mu \frac{k^2}{\varepsilon},$$

$$C_\mu = 0.09, \sigma_k = 0.82 \quad (5)$$

The dissipation tensor is defined from ε as:

$$\varepsilon_{ij} = (2/3) \delta_{ij} \rho \varepsilon \quad (6)$$

The dissipation rate (i.e. ε) in the above equation is defined by same equations (with ($\sigma_\varepsilon = 1.0$)) as in SKE.

The main difference between RSM models is the handling of pressure strain (ϕ_{ij}) and many different ways have been proposed for high- and low- Re versions [27-28, 31, 39-41]. The current work tests low- and high-Re versions of the Linear Pressure Strain (LPS) model and low-Re stress omega model formulations [30-31]. The high-Re version of LPS is used with SWF and NEWF. The low-Re version of LPS is used in conjunction with EWT.

Near-Wall treatment

Near-wall treatment is very important in wall-bounded turbulent flows. Walls have high velocity gradients and thus are the main source of turbulence production. These wall regions are differently handled in different models. The low-Re k- ε models (i.e. Abid, LB, LS, YS, AKN, CHC, RSM-S ω with low Re-correction) use damping functions and need a fine grid to integrate up to viscous sublayer ($y^+ = y u_\tau / \nu (\tau_w = \rho u_\tau^2) < 1$) [42]. In high-Re k- ε models (i.e. RKE, SKE, RNG etc.), the near-wall region is usually handled in two ways [30-33]: i) wall function approach without resolving the buffer and the viscous sublayers (applicable for $30 < y^+ < 300$: SWF and NEWF), ii) Two-layer model for ε and turbulent viscosity with single blended law of wall for mean velocity (EWT). Formulations for the different wall treatment methods (SWF, NEWF and EWT) are given elsewhere [30, 32-34, 43-44].

RSM model needs boundary conditions for Reynolds stresses in addition to the above wall treatment procedures. With SWF and NEWF, TKE is calculated using $k = 0.5 \overline{u'_i u'_i}$ away from the wall and in the near wall cells, a transport equation, similar to as in SKE, for TKE (with $\sigma_k = 0.82$) is solved with $\partial k / \partial n = 0$ at the wall. Afterwards, the individual Reynolds stresses are calculated using equation given below in near wall cells (derived based upon equilibrium of Reynolds stresses, i.e. production=dissipation) [30].

$$\overline{u'_t u'_t} / k = 1.098, \overline{u'_\eta u'_\eta} / k = 0.247, \overline{u'_\lambda u'_\lambda} / k = 0.655, \\ -\overline{u'_t u'_\eta} / k = 0.255 \quad (7)$$

Where, subscript t , η and λ stands for local tangential, normal and binormal coordinates respectively. With EWT, the normal derivatives of Reynolds stresses are taken zero at the wall.

MHD formulations

When the Magnetic Reynolds number, $Re_m = |\vec{v}| L (\mu_0 \sigma)$, is < 1 (such as for liquid metals), the induced magnetic field is negligible relative to the applied field. Based on Ohm's law and conservation of charge, coupled equations for electric potential, ϕ , and Lorentz force, \vec{F}_L can be solved as follows [45, 30].

$$\nabla^2 \phi = \nabla \cdot (\vec{v} \times \vec{B}_0) \text{ and } \vec{F}_L = \sigma (-\nabla \phi + \vec{v} \times \vec{B}_0) \times \vec{B}_0 \quad (8)$$

In time varying fields, and when the induced current is significant, (i.e. $Re_m > 1$), the Maxwell's equations are combined with Ohm's law to obtain a transport equation for the induced magnetic field, \vec{b} [45, 30].

$$\frac{\partial \vec{b}}{\partial t} + (\vec{v} \cdot \nabla) \vec{b} = \frac{1}{\mu_0 \sigma} \nabla^2 \vec{b} + ((\vec{B}_0 + \vec{b}) \cdot \nabla) \vec{v} - (\vec{v} \cdot \nabla) \vec{B}_0 - \frac{\partial \vec{B}_0}{\partial t} + \frac{1}{\mu_0 \sigma} \nabla^2 \vec{B}_0 \quad (9)$$

$$\vec{B} = \vec{B}_0 + \vec{b}; \quad \vec{J} = \nabla \times \vec{B} / \mu_0; \quad \vec{F}_L = \vec{J} \times \vec{B}$$

In both above methods, the Lorentz force is applied as a source term in the momentum equations.

Effect of magnetic field on turbulence in RANS turbulence models

Many researchers [12-15, 46] improved the conventional non-MHD RANS turbulence models for the effect of the magnetic field on the turbulence in low magnetic Reynolds number liquid metal MHD flows. Ji and Gardner [14] proposed and tested source terms for magnetic field damping effects on turbulence using a k- ε model on a turbulent conducting liquid flow in an insulated pipe. Velocity profiles, skin friction, temperature profiles, Nusselt numbers showed agreement with available experimental data for range of Re and Ha. The biggest shortcoming of this model was the usage of bulk Stuart number (or interaction parameter, Ha^2/Re) to define the turbulence damping terms making it a bulk flow dependent model and only applicable in the standard problems where bulk Stuart number can be easily defined. Smolentsev et al [15] proposed different source terms for k- ε models but again based up on the bulk flow Stuart number. The model was found to match experiments closely in free surface channel flow.

Galperin [46] proposed a second-moment closure model for MHD turbulence, although this model was not numerically tested on conventional flows. Kenjereš and Hanjalić [12-13] proposed new source terms for k- ε and second-moment closure models (RSM). The improved k- ε model was validated with the DNS results in a channel flow under transverse magnetic field. After validation, the model was used in a 3-d developing rectangular duct flow with partial magnetic field and model was found performing well for mean velocities. No assessment for turbulence parameters was made in rectangular duct flow. Kenjereš and Hanjalić [12-13] also proposed a similar closure for $\overline{u'_i u'_j}$ equations for MHD effects in RSM as proposed by Galperin [46]. This closure for RSM showed considerable improvement of results in a channel flow. The current study includes the models proposed by Kenjereš and Hanjalić's [12-13] for the channel

and square duct flows. The following modifications were made to the models.

k-ε model :

$$k\text{-equation: } S_k^M = -\sigma B_0^2 k \exp(-C_1^M (\sigma / \rho) B_0^2 (k / \varepsilon)) \quad (10)$$

$$\varepsilon\text{-equation: } S_\varepsilon^M = -\sigma B_0^2 \varepsilon \exp(-C_1^M (\sigma / \rho) B_0^2 (k / \varepsilon)) \quad (11)$$

where, $C_1^M = 0.025$

Reynolds Stress Model (RSM) MHD source terms:

After simplification for y-directional (vertical) magnetic field and some algebra the six independent Reynolds stress transport equations can be derived with the following MHD source terms;

$$\overline{w'w'}\text{-equation: } S_{w'w'}^M = \sigma (-2B_{y0} \overline{w'\partial\phi'/\partial x} - 2B_{y0}^2 \overline{w'w'}) \quad (12)$$

$$\overline{v'v'}\text{-equation: } S_{v'v'}^M = 0 \quad (13)$$

$$\overline{u'u'}\text{-equation: } S_{u'u'}^M = \sigma (2B_{y0} \overline{u'\partial\phi'/\partial z} - 2B_{y0}^2 \overline{u'u'}) \quad (14)$$

$$\overline{u'v'}\text{-equation: } S_{u'v'}^M = \sigma (B_{y0} \overline{v'\partial\phi'/\partial z} - B_{y0}^2 \overline{u'v'}) \quad (15)$$

$$\overline{w'u'}\text{-equation: } S_{w'u'}^M = \sigma (-B_{y0} \overline{u'\partial\phi'/\partial x} + B_{y0} \overline{w'\partial\phi'/\partial z} - 2B_{y0}^2 \overline{w'u'}) \quad (16)$$

$$\overline{w'v'}\text{-equation: } S_{w'v'}^M = \sigma (-B_{y0} \overline{v'\partial\phi'/\partial x} - B_{y0}^2 \overline{w'v'}) \quad (17)$$

Source term for scalar dissipation rate (ε) is defined as [13];

$$S_\varepsilon^M = 0.5 S_{ii}^M \varepsilon / k \quad (18)$$

It can be seen that all the source terms due to the magnetic field are negatively correlated with the corresponding Reynolds stress therefore sinks to the Reynolds stresses. It is interesting to note that the magnetic field causes no direct sink to the Reynolds normal stress parallel to magnetic field (i.e. $\overline{v'v'}$). The indirect suppression effect on $\overline{v'v'}$ is via Reynolds shear stresses. In the above sinks, the terms involving correlation of velocity fluctuation with electric potential gradient require modeling and cannot be incorporated directly in RSM. Kovner and Levin [47] suggested a way to model electric potential-velocity correlation. Galperin [46] and later Kenjereš and Hanjalić [12-13] followed their method and came up with following formulation for the correlation;

$$\partial\phi'/\partial x_k = \beta \varepsilon_{kmn} \overline{u'_m B_{n0}} \Rightarrow \overline{u'_i \partial\phi'/\partial x_k} = \beta \varepsilon_{kmn} \overline{u'_i u'_m B_{n0}} \quad (19)$$

Galperin [46] proposed $0 < \beta < 1$. Kenjereš and Hanjalić [13] proposed $\beta = 0.6$ via MHD channel flow. In the current work, the value of β as proposed by Kenjereš and Hanjalić is used.

The above discussed two formulations for k-ε and RSM for the effect of magnetic field on turbulence have been implemented using a UDF with the magnetic induction and the electric potential methods [30]. More details on various turbulence models, wall treatment approaches, magnetic induction and electric potential method for MHD calculations can be found in [30].

DNS DATABASES

Five DNS databases were used to assess the above models. The conditions for various DNS databases are given in Table 1.

High-Reynolds Number Non-MHD channel flow

Satake et al [48] performed DNS calculations in a non-MHD channel at a bulk Reynolds number of ~45818 using 800 million nodes. The mean velocities, RMS of velocity fluctuations and TKE budgets were reported. This non-MHD case was used as a base case to first evaluate the purely hydrodynamic models.

Table 1 Various parameters in different DNS calculations considered during evaluation [48-52]

Geom.	Re	Grid (N _x xN _y xN _z) Domain (XxYxZ)	Ha	W _b / $d\bar{p}/dz$
Channel (Case-1)	45818 (Re _τ =1120) (Satake et al)	1024x1024x768 $\pi x 1 x 2.5\pi$	0	20.45 /2.0
Channel (Case-2)	4586 (Re _τ =150) (Iwamoto et al)	128x97x128 $\pi x 1 x 2.5\pi$	0	15.28 /2.0
Channel (Case-3)	4710 (Re _τ =150) (Noguchi et al)	64x128x64 0.5 $\pi x 1 x 1.25\pi$	6. 0	15.7 /2.0
Square duct (Case-4)	5466 (Re _τ =360) (Shinn et al)	160x160x1024 1x1x8	0	15.18 /4.0
Square duct (Case-5)	5602 (Re _τ =361) (Chaudhary et al)	128x128x512 1x1x16	21 .2	1.057 /0.018

Where, $Re_\tau = \frac{D_1 u_\tau}{\nu}$, $Re = \frac{D_2 W_b}{\nu}$, and $Ha = B_{y0} D_1 \sqrt{\frac{\sigma}{\rho \nu}}$.

Channel: $D_1 = \delta$, $D_2 = 2\delta$ ($\delta = 0.5$ is half channel height)

Square duct: $D_1 = D_2 = D$, ($D = 1$ is the side of the square duct)

Low-Reynolds Number MHD and Non-MHD channel flows

The non-MHD channel flow data of Iwamoto et al [49] has been used to test performance of RANS models at lower Reynolds numbers. In his case, $Re_\tau (= \delta u_\tau / \nu) = 150$, corresponding to bulk $Re (= 2\delta W_b / \nu)$, δ : half channel height) = 4586 was used. To test the models for MHD turbulence, the MHD channel case of Noguchi et al [50] ($Re_\tau (= \delta u_\tau / \nu) = 150$, bulk $Re (= 2\delta W_b / \nu) = 4710$, $Ha (= \sqrt{\sigma / (\rho \nu)} B_0 \delta) = 6$, δ : half channel height) was used.

Low-Reynolds Number MHD and Non-MHD square duct flows

A GPU based code (CU-FLOW) [51] that has been previously used for DNS calculations in a non-MHD square duct has been extended for DNS calculations of a MHD square duct [52]. For the non-MHD case, ($Re_\tau (= D u_\tau / \nu) = 360$, bulk $Re (= D W_b / \nu) = 5466$), a duct of size of 1x1x8 non-dimensional units and 160x160x1024 control volumes (with 1% grid stretching in wall normal directions) were used. For the MHD case, ($Re_\tau (= D u_\tau / \nu) = 361$, bulk $Re (= D W_b / \nu) = 5602$, $Ha (= \sqrt{\sigma / (\rho \nu)} B_0 D) = 21.2$)) a duct of size of 1x1x16 non-dimensional units with 128x128x512 control volumes (with 2% grid stretching in wall normal directions) were used. Both these simulations were shown to give grid-independent solutions to the relevant equations.

COMPUTATIONAL DETAILS

Computational Domain, Boundary Conditions and Numerical Method

Taking advantage of fully-developed flow with RANS models, the domain size was taken as 1x1x1 non-dimensional units for both the channel and the square duct. For the channel, the top and the bottom walls were electrically

insulated with no-slip velocity conditions while the streamwise (z-) and spanwise (x-) directions were considered periodic. In the square duct, the four walls (top, bottom, right and left) were electrically insulated with no-slip velocity conditions whereas the streamwise direction (z-) is periodic. For the MHD calculations, the magnetic field was applied in the vertical (y-) direction. The simulations were carried out by fixing the bulk mean flow Reynolds number as given in Table 1 with the mean streamwise pressure gradient free to change. All the calculations were performed using FLUENT's steady-state segregated solver with SIMPLE algorithm for pressure-velocity coupling with either magnetic induction or electric potential methods for MHD calculations [30]. For each case, the results were ensured to be grid-independent by systematically increasing the number of control volumes until a grid-independent solution is obtained. All cases were converged such that the unscaled absolute residuals reached below 10^{-3} to stagnant values.

Grids

For the high-Re calculations (case 1, $Re=45818$) with EWT, five grids with ten control volumes each in streamwise (z-) and spanwise (x-) directions were used. In the wall-normal (y-) direction, three uniform grids (consisting of 50, 80 and 130 control volumes) and two non-uniform grids (near-wall $y^+ = 1$) were used. Figure 1 compares the TKE along the wall normal direction in the case of the RKE model with EWT. The results show grid independence as y^+ approached a value of one in the cells adjacent to the wall. The coarse grids produced peaks in k near the wall that appear closer to the true DNS solution. This occurs if the cell next to the wall is in the buffer region for the models with EWT. However, the trend is better-matched with the fine grids. Similar behavior was seen for the other high-Re models (RNG, SKE and RSM-LPS); hence grid independence plots for other models are not presented. All models obtained grid independence with a 139(non-uniform)x10x10 grid, so this grid was used for evaluation of these models. For the models using the SWF and NEWF approaches, the first cell center next to the wall should be placed in the range of $30 \leq y^+ \leq 300$ and, arbitrary grid refinement close to the wall is not appropriate. Hence, only uniform grids of 30x10x10 with y^+ in cells next to the wall being in the range of 35-40 are used for models with these wall functions.

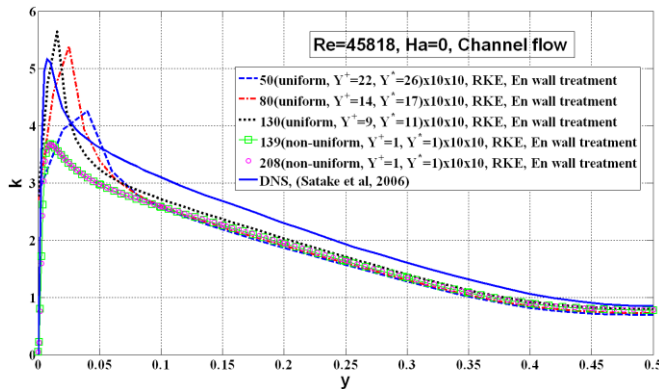


Figure 1 Grid independence study in high-Re channel flow for RKE with EWT

For low-Reynolds number flows (cases 2-5), the number of cells required to satisfy near-wall $y^+ > 30$ is too small to be accurate. Hence, SWF and NEWF were not evaluated for low-Re flows. Only low-Re models (Abid, LB, LS, YS, AKN, and CHC) or high-Re models (like SKE, RNG (with low-Re differential viscosity model), RKE, and RSM-linear pressure-strain) with EWT are considered. Two uniform (50x10x10 and 80x10x10) and one non-uniform (100x10x10)

grids were used for RKE, SKE, RNG, and RSM-LPS models with EWT to ascertain grid independency. The same grids were also used for the RSM-S ω (with low-Re correction) model. As the grid is refined to 100 non-uniformly-spaced cells, the results show very good grid independence (mostly for all above models). Hence this grid is used in all subsequent computations of low-Re cases with these models. For the square duct, the same grid is used in both the wall-normal directions (i.e. 100 x 100 x 10 cells).

Grid-convergence tests were also systematically done for each of the six low-Re k- ϵ models. All low-Re k- ϵ models were observed to achieve grid independence with 120 cells in the wall normal direction (giving a near-wall y^+ between 0.55-0.9). Hence this grid is used in all subsequent computations of low-Re cases with these models. In square duct flows, the same grid resolution of 120 cells is used in both wall-normal directions (i.e. 120x120x10).

Computational Costs

Due to their varying complexities and convergence rates, both the total and per-iteration computational times for each model were different. The time per iteration and total number of iterations to final convergence required by FLUENT (using 6 cores of a Dell Precision T7400 workstation with 2.66 GHz Intel Xeon processor and 8 GB RAM) with different models is discussed here based on calculations. As expected, the two equation models RKE, RNG and SKE with EWT require nearly the same time (per iteration as well as total time). On a per-iteration basis, the various two equations models are 5-30% less expensive than RSM-LPS (which solves 7 transport equations) with EWT. However, to obtain final converged results, RSM-LPS model is ~13-26 times more expensive. With SWF and NEWF, the two equation models are about 20-30% less expensive than RSM-LPS when compared on a per iteration basis but the time required to final convergence by RSM-LPS model reduces and it is only slightly more expensive. It seems that with finer grids, RSM-LPS model becomes increasingly expensive to achieve final convergence relative to two equation models. The EWT and SWF/NEWF are almost equally expensive for the same grid, but the grid required for EWT is much higher. In all models tested, the computational requirement increases almost linearly with the grid size. Surprisingly, low-Re RSM-S ω model, which also solves 7 equations, is only about twice as expensive as the two equation models. All low-Re k- ϵ models take nearly the same time per iteration, but the total times for LB and LS models are smaller. YS model took five times more time than LB and LS.

RESULTS AND DISCUSSION

Results are first presented for non-MHD flows to show the accuracy of the various models without magnetic field. From these, models giving the best agreement are evaluated for the MHD flows after incorporating the changes due to the magnetic field effects.

High-Reynolds number non-MHD channel flow (Re=45818)

Figure 2 compares the TKE predicted by the various models with the DNS data of Satake et al. [48] for the grid independent mesh with EWT. It is seen that all models (RKE, RNG, SKE, and RSM-LPS) give nearly the same distribution of the TKE. They underestimate the DNS peak values near the wall by 22-27%. Error decreases with distance from the wall, and TKE in the central core is predicted within 10%. Figure 3 shows similar behavior with SWF. As theoretically

required, the near-wall y^+ has been maintained around 36-37. The results with SWF were nearly the same as with the NEWF probably because of the lack of flow separation or pressure gradient effects in a channel flow. As seen with the EWT, the peak value of TKE was again under-predicted, this time by a larger amount (42%). The agreement in the core region is much better with all the models, except RKE giving slightly lower predictions.

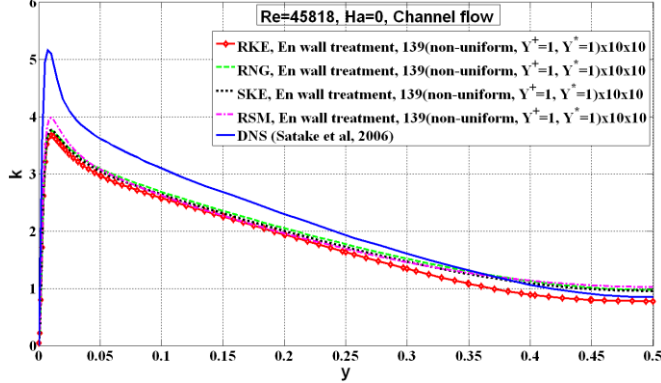


Figure 2 Comparison of TKE in various models with EWT in high-Re channel flow

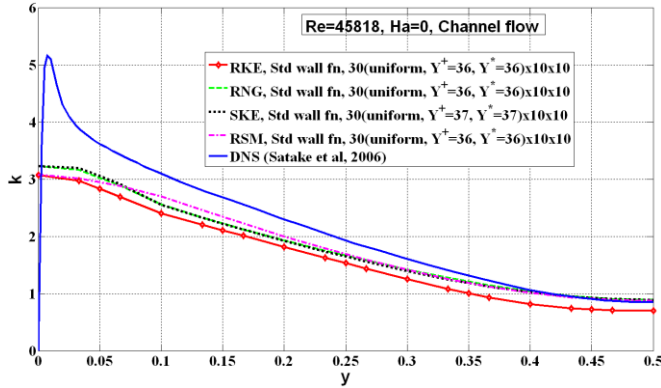


Figure 3 Comparison of TKE in various models with SWF approach in high-Re channel flow

The non-dimensionalized mean axial velocities predicted with the SKE and RSM-LPS models using EWT and SWF are presented in Figure 4. The velocity profiles with NEWF are not presented as they were nearly the same as with SWF. It is seen that the EWT with $y^+=1$ resolves velocity accurately all the way up to the viscous sublayer and matches best with the DNS results across the whole channel. Both models performed equally well with EWT, with errors consistently within 3%. With SWF, as y^+ is maintained ~ 36 , the cell next to the wall stays in log-law region. Again both models predicted mean velocities well, although error with the RSM-LPS model increased to $\sim 5\%$ in the central core.

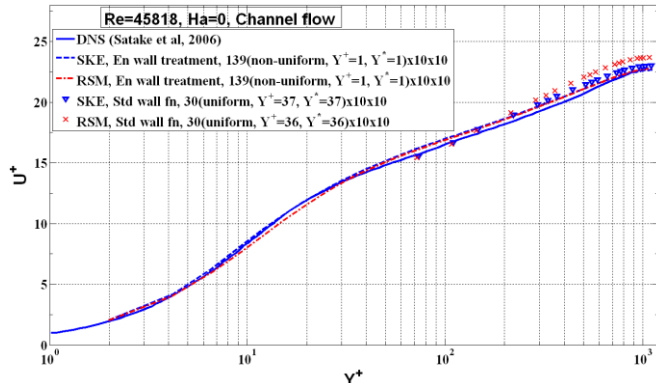


Figure 4 Comparison of normalized mean axial velocity in SKE and RSM-LPS with SWF and EWT in high-Re channel flow

The Reynolds normal stresses predicted by the RSM-LPS model with EWT are compared with the DNS data in Figure 5. Comparison with SWF and NEWF is not presented here. However, with SWF and NEWF, the predictions matched closely with the DNS data in the core region except for the wall normal velocity fluctuations, which were underpredicted. The errors increased towards the wall especially in the axial and wall normal velocity fluctuations. Both wall functions performed equally but both missed the peak values close to the wall in all the three velocity fluctuations. The peak value of the RMS of axial velocity fluctuations is underpredicted by $\sim 36\%$ while the error in transverse and spanwise velocity fluctuations is smaller. The RMS of spanwise velocity fluctuations matched best with the DNS. The RSM-LPS model with EWT performed better than with SWF or NEWF in predicting all three velocity fluctuations, as expected. Again, the spanwise velocity fluctuations were predicted most accurately followed by wall normal fluctuations. The error in predicting peak value of axial velocity fluctuations reduced from $\sim 36\%$ to $\sim 12\%$ by using the EWT. Overall, RSM-LPS with EWT predicted the anisotropy of Reynolds normal stresses reasonably well.

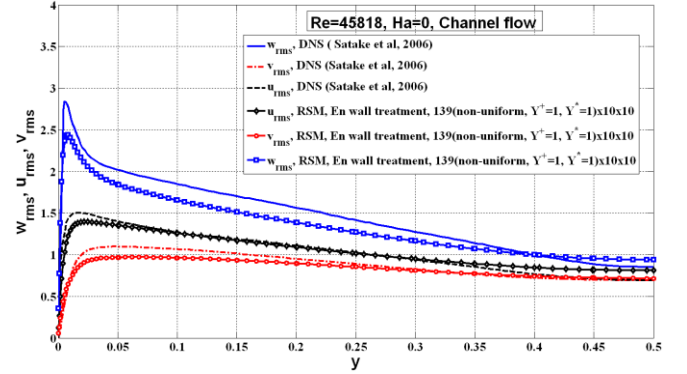


Figure 5 Comparison of RMS of velocity fluctuations in RSM-linear-pressure-strain with EWT in high-Re channel flow

Low-Reynolds number non-MHD channel flow (Re=4586)

We next consider the low-Re non-MHD channel flow for which the various low-Re turbulence models are first evaluated. Figure 6 compares the TKE predicted by various low-Re $k-\epsilon$ models with the DNS.

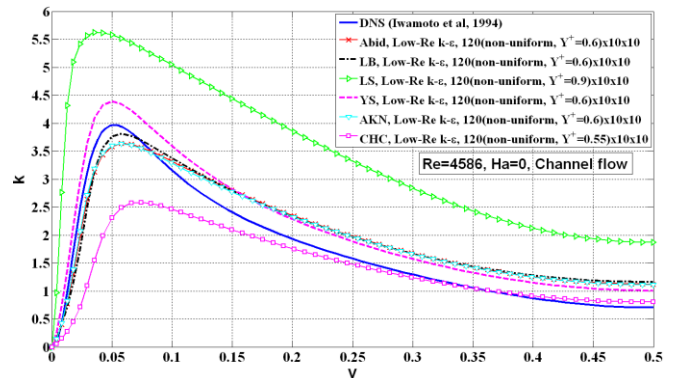


Figure 6 Comparison of TKE predicted by low-Re $k-\epsilon$ models with the DNS in low-Re channel flow

The LS model greatly overpredicted throughout the domain, while the CHC model underpredicted near the wall and matched in the core. The 4 remaining models predicted similar values, matching the DNS data within 15% error near the wall but over-predicting (by $\sim 60\%$) in the core. Overall, the LB model performed the best of all the models. The YS model gave the correct trend across the whole domain, consistently overpredicting by 7-30%. The best low-Re $k-\epsilon$

models (LB, AKN, and YS) are evaluated for mean axial velocity predictions. All three models predicted the mean axial velocity profile across the channel very well (within 5% error).

In addition to the low-Re $k-\epsilon$ models, the high-Re $k-\epsilon$ models with EWT (RKE, RNG with differential viscosity, and SKE) and RSM models (RSM-LPS with EWT and RSM-S ω low-Re) also have been evaluated in this low-Re non-MHD channel flow. All models, except RNG and RSM-S ω , performed similarly by matching the peak values but overpredicting the values significantly (by $\sim 120\%$) in the core. The RNG model overpredicted slightly more in the core than other models. RSM-S ω model matched TKE better in the core. To understand the turbulence anisotropy capture by RSM models, the RMS of velocity fluctuations predicted by low-Re RSM-S ω and RSM-LPS model were compared (not shown here) with the DNS. The RSM-S ω model, although it predicted the TKE best in the core, did not capture the anisotropy of Reynolds stresses even qualitatively. Because it was outperformed by the RSM-LPS model, the RSM-S ω model was not considered further in this study. The RSM-LPS model with EWT captured anisotropy qualitatively in all velocity fluctuations but overpredicted in the core. Figure 7 shows the comparison of the mean axial velocities given by RKE, SKE, and RSM-LPS models. All matched the DNS data closely except for some underprediction in the core.

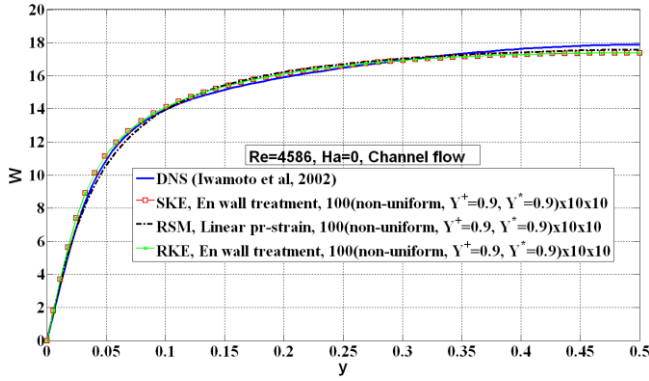


Figure 7 Comparison of mean axial velocity by SKE, RKE, RSM-LPS models with EWT with the DNS in low-Re channel flow

Low-Reynolds number MHD channel flow (Re=4710, Ha=6)

The models (LB, SKE, and RSM-LPS) which performed better in low-Re non-MHD channel flow were then tested in low-Re MHD channel flow at a Reynolds number of 4710 and Ha = 6.0. Comparison of the computed TKE using the selected turbulence models with and without inclusion of the MHD sources/sinks is shown in Figure 8.

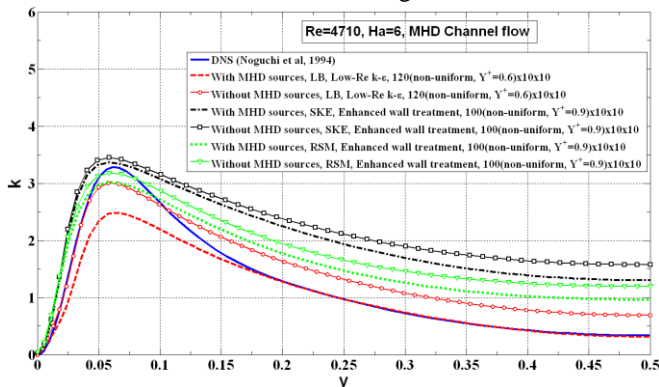


Figure 8 Comparison of TKE in low-Re MHD channel flow with various models

The LB low-Re $k-\epsilon$ model with MHD sources/sinks matches the DNS computed turbulent kinetic energy quite well in the core but underpredicts the high values close to the wall calculated by the DNS. This match in the core seems to be fortuitous when overall trend is not predicted that well. The peak TKE is seen to be better predicted by LB without the MHD sources. The effect of the MHD sources/sinks on suppressing turbulence is clearly seen. SKE and RSM with EWT matched the peak values closely but overpredicted greatly (by 300-500%) in the core. The models using EWT show very little effect of MHD source terms. This is likely due to the lack of magnetic field effects in wall treatment method. This contrasts with the strong effect observed in the low-Re LB model, where the source terms are applied throughout the domain.

Figure 9 compares the axial velocity in wall coordinates. The LB low-Re $k-\epsilon$ model with MHD sources gives the best agreement with DNS data. However, part of profile in between $15 < y^+ < 80$ is under-predicted. The second best prediction is from the LB model without MHD sources. The predictions of RSM and SKE are similar, with the RSM-LPS performing slightly better. The underprediction of the normalized velocity in the core is mainly due to the higher frictional losses leading to higher friction velocity. The SKE and RSM models with EWT do not show much effect of MHD sources in the mean velocity. Figure 10 compares the axial velocity, as in Figure 9, but this time non-normalized mean velocity as a function of distance from the wall in the wall normal direction. The close match of predictions from all models with the DNS reinforces the assertion that the higher frictional losses are causing the differences in predictions in Figure 9.

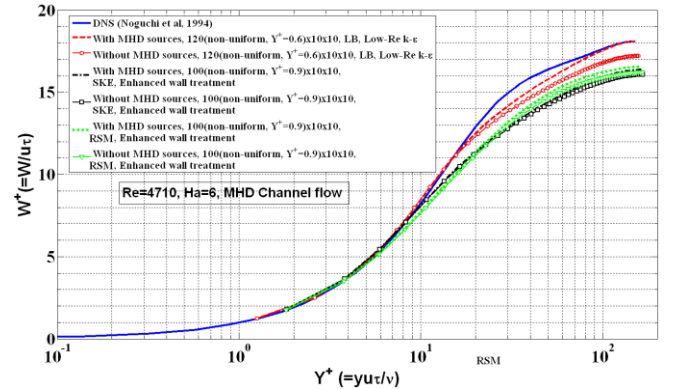


Figure 9 Comparison of normalized mean axial velocity vs. normalized wall distance in wall units in low-Re MHD channel flow in various models

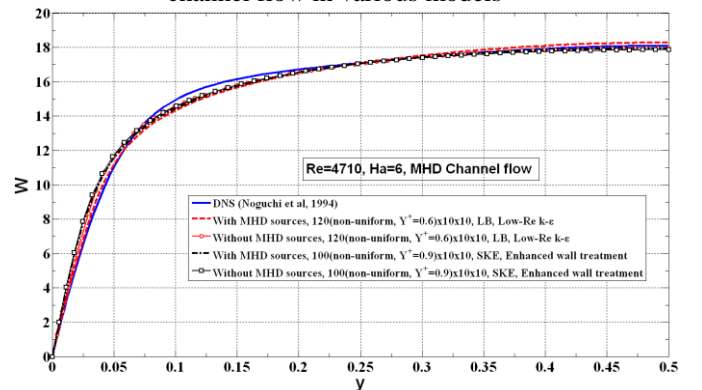


Figure 10 Comparison of mean axial velocity vs. distance from the wall in low-Re MHD channel flow in LB and SKE models

We next examine the MHD source/sink terms in the k -equation and compare their magnitude with those extracted from the DNS budgets (Figure 11). The trends predicted by all

3 models are reasonable, but the LB low-Re k- ϵ model matches best with the DNS (within 20%). Although, the SKE model predicts the peak closely, it overpredicts the values in the core by ~300%. Interestingly, none of the models capture the small positive peak very close to the wall.

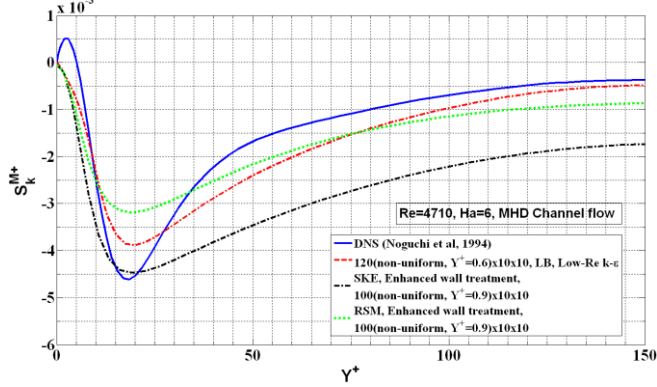


Figure 11 Comparison of the MHD source/sink in the k-equation / budget (DNS) in low-Re MHD channel flow in various models with the DNS

Figure 12 presents the sink term due to magnetic field in the turbulent dissipation rate (ϵ) equation. All 3 models correctly predict the asymptotic decay of source to dissipation to zero in the core. The LB low-Re model correctly predicts the profile qualitatively across the whole channel but underestimates the values. The SKE and RSM models predict qualitatively similar profiles with negative peaks at $y^+ \sim 10$. The SKE model gives the closest match although errors approach 50% near the wall.

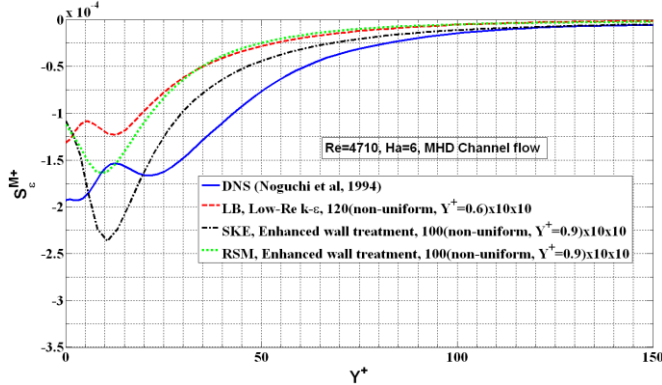


Figure 12 Comparison of MHD sink in ϵ -equation / budget (DNS) in low-Re MHD channel flow in various models with the DNS

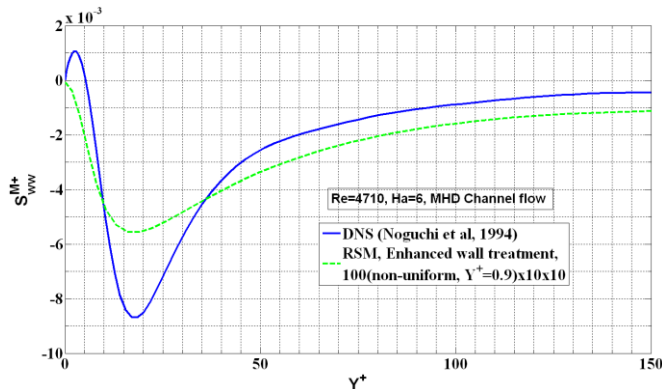


Figure 13 Comparison of the MHD source/sink in $w'w'$ -equation / budget (DNS) in low-Re MHD channel flow in RSM-LPS model with the DNS

Figures 13 and 14 give comparisons of the magnetic field source/sink terms in Reynolds normal stresses obtained by RSM-LPS. For $S_{w'w'}^{M+}$, RSM behaves similar to the TKE source. It underpredicts the peak value and overpredicts in the core. The positive values, which indicate a source in $S_{w'w'}^{M+}$

below $y^+ < 5$, are again missed by the model. The MHD sink in S_{uu}^{M+} is qualitatively captured but the values are over-predicted across the whole length.

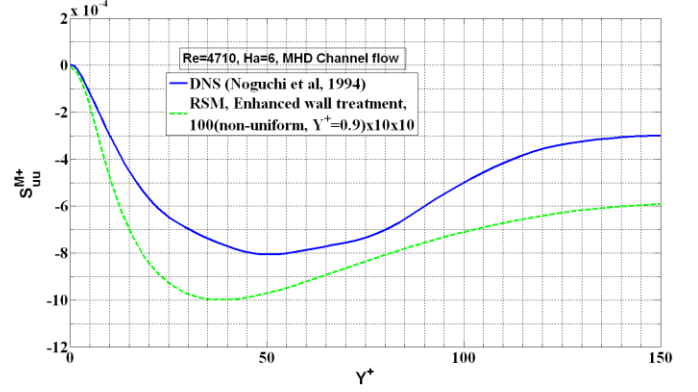


Figure 14 Comparison of MHD source/sink in $\overline{u'u'}$ -equation / budget (DNS) in low-Re MHD channel flow in RSM-LPS model with DNS

Low-Reynolds number non-MHD square duct flow (Re=5466)

The models are next evaluated for the fully-developed turbulent flow in a square duct bounded by four walls. For this case, it is well-known that the anisotropy in the Reynolds stresses generates cross-stream flows [16], which are not present in the laminar case. Turbulence models based on isotropic eddy-viscosity cannot predict such secondary flows [16]. To predict the secondary flows, it is necessary to use either non-linear/anisotropic two equation models [53-56], RSMs [57], or algebraic stress models [58]. Hence, models other than the above are not expected to be accurate. However, they have been considered in this study to assess their inaccuracy and to evaluate their relative performance against the more expensive RSM. Figure 15 presents the comparison of TKE along vertical bisector in a non-MHD square duct using LB, RKE, SKE and RSM-LPS models. The grid in all models resolved the flow up to the viscous sublayer ($y^+ \sim 1$). The LB model predicts the TKE better than other models. However, all models give excessive TKE in the core region by over 100%.

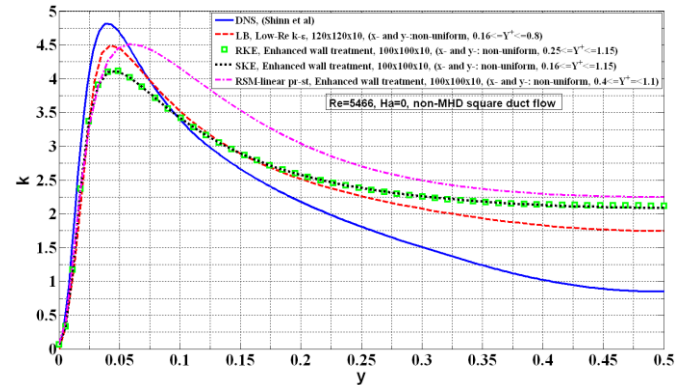


Figure 15 Comparison of TKE predicted by various models with the DNS along vertical bisector in a non-MHD square duct

Figure 16 compares the mean axial velocity along the vertical bisector obtained by the different models. The RKE, SKE and LB models show similar reasonable behavior, as they agree with the DNS within ~8%. All 3 models overpredict in-between the wall and the core and underpredict in the core region. The RSM model expectedly is slightly better but matches the other models in underpredicting the core region. Compared to the channel, the square duct flow is predicted with less accuracy, probably as a result of the inability to predict the secondary flows.

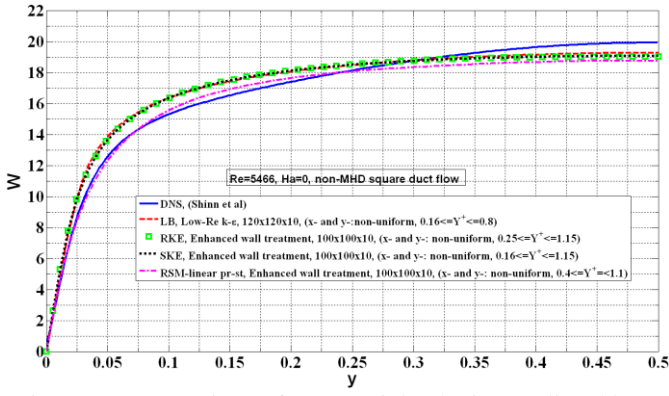
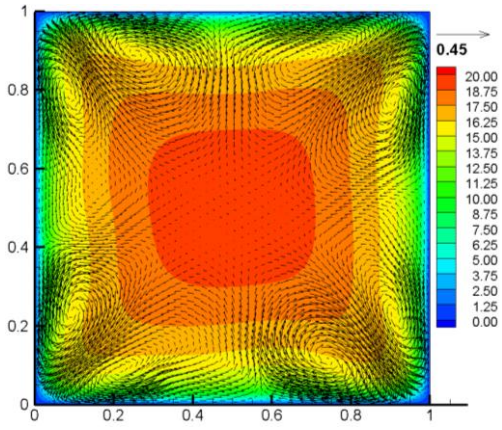
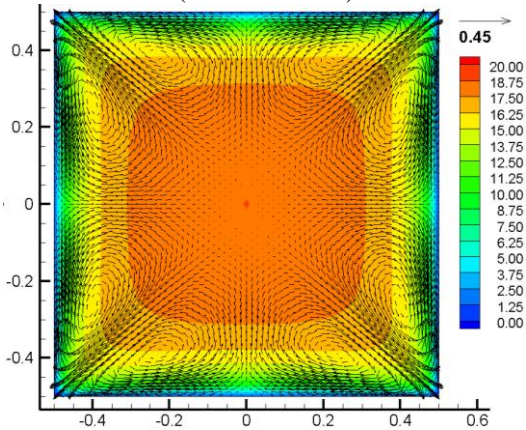


Figure 16 Comparison of mean axial velocity predicted by various models with the DNS in non-MHD square duct along vertical bisector

Figure 17(a) and (b) show the mean axial velocity contours and secondary velocity vectors obtained by the DNS and the RSM-LPS model. Only the RSM model predicts the secondary flows, and hence results of other models are not shown. The bulging of the axial velocity profile is not predicted to the extent observed in the DNS.



(a) DNS (Re=5466, Ha=0, Shinn et al [51]) (160x160x1024)



(b) RSM-linear-pr-strain, En wall treatment (Re=5466, Ha=0, 100x100x10)

Figure 17 Comparison of mean axial velocity contours and secondary velocity vectors in non-MHD square duct

Low-Reynolds number MHD square duct flow (Re=5602, Ha=21.2)

The final test case considered is the MHD square duct flow, which is an appropriate geometry for the industrial application of electromagnetics to control flow in the continuous casting of steel. Here again, both isotropic

viscosity-based models and the RSM models are evaluated, realizing still that the former cannot predict even qualitatively the cross-stream flow fields. Because of the magnetic field effects, for a square duct, the profiles of various quantities differ between the vertical and the horizontal bisectors. Hence profiles are compared along both these directions. Although the calculations of the channel flow were performed only using the magnetic induction method available in FLUENT, in the square duct flow, both magnetic induction and electric potential methods have been tested. The maximum magnitude of the induced magnetic field in the current simulations is only 0.039% of the externally applied field, so the magnetic induction method and electric potential method give virtually identical results.

Figures 18 and 19 compare the TKE along vertical and horizontal bisectors respectively obtained from various models and the results of the DNS.

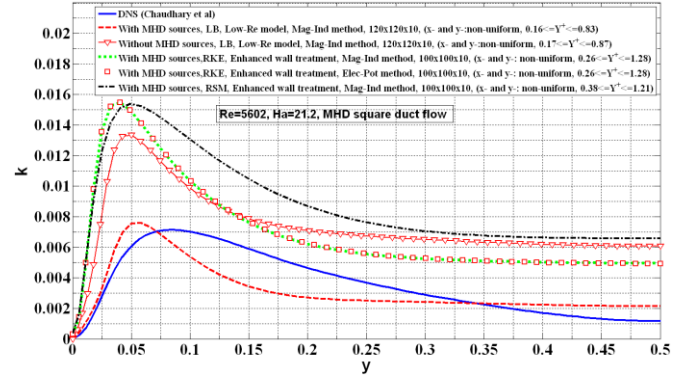


Figure 18 Comparison of TKE in various models with the DNS in MHD square duct along vertical bisector

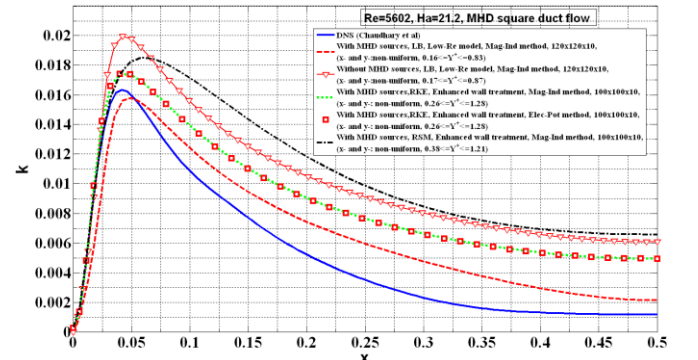


Figure 19 Comparison of TKE in various models with the DNS in MHD square duct along horizontal bisector

It can be seen that MHD suppresses the turbulence more along the vertical bisector than along the horizontal bisector and only the LB model with MHD sources is able to predict this trend, reasonably matching with DNS (generally within 50%). The results with LB without MHD sources overpredict the DNS data by 100-500%. The MHD sources/sinks proposed by Kenjereš and Hanjalić [12-13] provide significant improvements by predicting the correct trend of turbulence suppression, especially using the LB model. Both the RKE and RSM models over-predict the turbulence energy in the core along both the bisectors by ~500%. Moreover they do not capture the strong differential suppression of turbulence along the two bisectors, as was seen in the DNS and in the results of LB model with MHD sources. On the horizontal bisector close to the side walls, turbulence is not suppressed much because the induced current is parallel to the magnetic field in this region. The RKE and RSM models predict the peak value of the TKE better along the horizontal bisector. Surprisingly, the RSM model is found to perform the

worst among the tested models for suppressing turbulence by magnetic field effects.

Figure 20 presents mean axial velocity contours and mean secondary velocity vectors in the cross-section. As shown by the DNS, the mean axial velocity contours and the secondary flows are significantly altered in the presence of the transverse magnetic field. The secondary velocities, rather going into corners, now go towards the top and bottom walls, thus lifting the axial velocity contours in these regions towards the top and the bottom walls. After hitting the walls, these secondary flows move parallel to the top and bottom walls before turning towards the core at the center and thus cause a strong bulging in mean axial velocity there. This effect of strong bulging is not seen close to the side walls. It can be seen that none of the models is able to capture this effect. Although RSM predicts secondary flows, the differential effect of the magnetic field close to the top /bottom walls and the right/left side walls is missing. RSM predicts almost symmetric mean secondary and axial velocities except for a slight elongation of mean axial velocity (i.e. flattening) in the vertical direction. As mentioned earlier, LB and RKE do not predict secondary flows at all and over-predict the velocity flattening in the vertical direction. Both the k- ϵ models (LB and RKE) predict similar axial velocity across the cross-section.

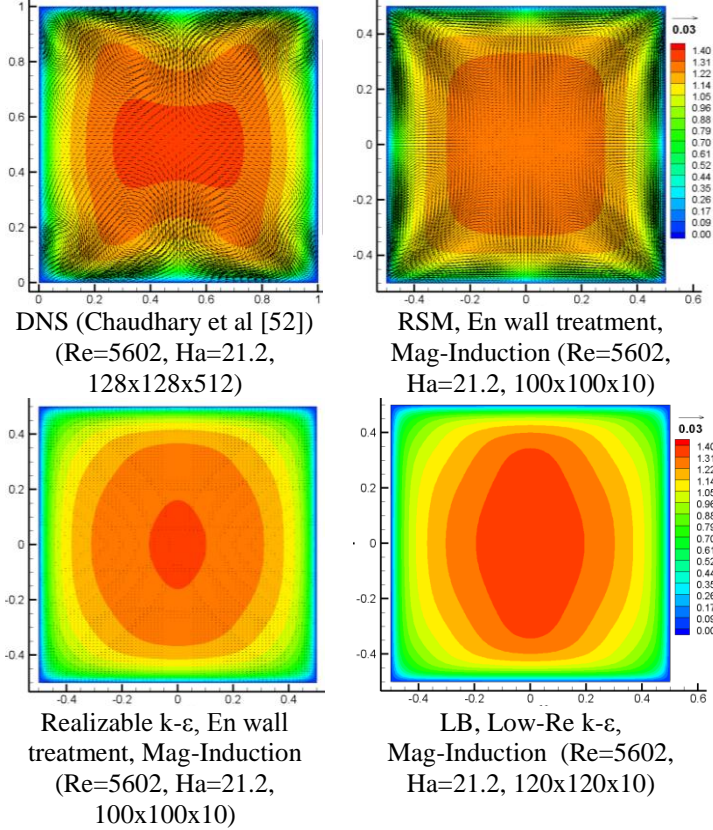


Figure 20 Comparison of mean axial velocity contours and secondary velocity vectors in MHD duct

Figure 21 and 22 show the MHD sources/sinks in the TKE equation computed by the various models. The velocity-electric potential gradient correlation acts as a source whereas the Reynolds normal stresses perpendicular to the magnetic field act as sinks, as shown in the DNS data. The sink is stronger than the source giving a net effect of suppressing the turbulence. It can be seen that the LB model predicts this source reasonably correctly, followed by RKE and then RSM-LPS. The predictions are better along the stronger Lorentz force bisector. Both the RKE and the RSM-LPS over-predict the MHD sources to TKE along both bisectors.

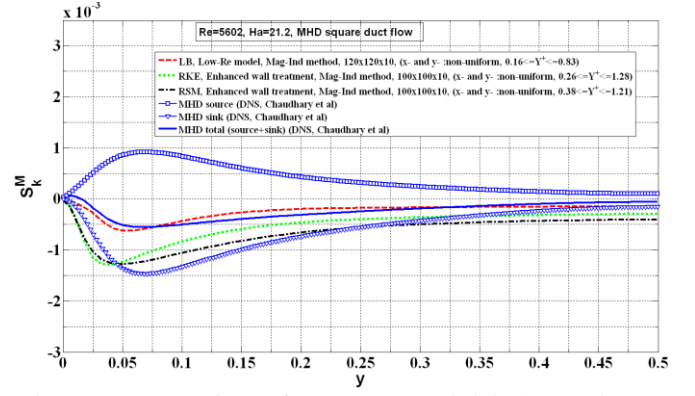


Figure 21 Comparison of MHD source/sink in k-equation / budget (DNS) predicted by various models with the DNS in MHD square duct along vertical bisector

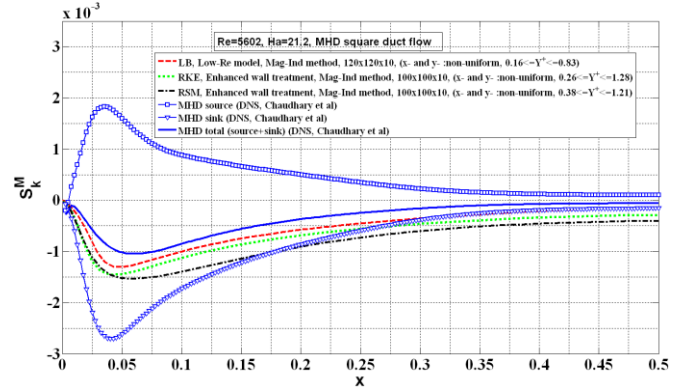


Figure 22 Comparison of MHD source/sink in k-equation / budget (DNS) in various models with the DNS in MHD square duct along horizontal bisector

The friction factor along bottom horizontal and left vertical walls is presented in Figure 23.

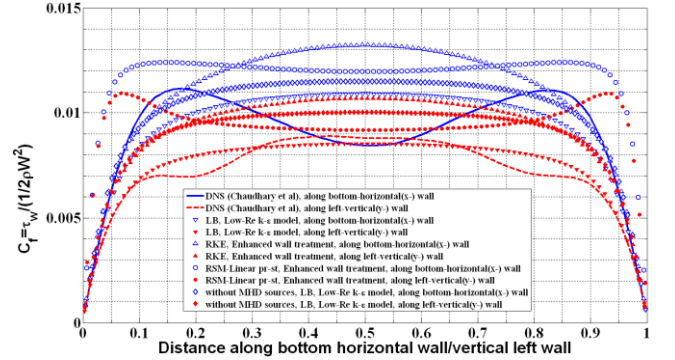


Figure 23 Comparison of the friction factor in MHD square duct along bottom-horizontal and left-vertical walls in various models with the DNS

Along the bottom horizontal wall, the friction factor shows two side peaks with a large dip at the center. Along left-vertical wall, the friction factor shows a central flat region with two side dips. None of the models is seen to predict these trends correctly. Both the k- ϵ models (LB and RKE) give similar profiles, with a central overpredicted peak. The RSM-LPS model predicts the side peaks with a central dip along both walls but does not completely agree with the DNS results. RSM suggests larger frictional losses, especially in the corners. The best agreement is seen with LB model with MHD sources. The LB model, without MHD sources, overpredicts friction along both walls.

CONCLUSIONS

In this study several turbulence models of k- ϵ and Reynolds stress transport category are evaluated for their

ability to predict turbulent flow fields subjected to a magnetic field. Five test cases of flows in a channel and square duct have been computed and the results are compared with DNS data. The MHD sources/sinks in k - and ε - equations for k - ε models and in Reynolds stresses for RSM, as proposed by Kenjereš and Hanjalić [12-13], were implemented through UDFs in the FLUENT code. The performance of these models, on the basis of their predictions of mean velocities, RMS of velocity fluctuations, TKE, MHD sources and frictional losses can be summarized as follows:

In both high- and low-Re channel flows, all of the models predicted mean axial velocity reasonably well (within 5% error), given fine-enough grids for grid-independence (EWT and low-Re) or satisfaction of the y^+ requirements (SWF and NEWF). However, the TKE was much less accurate, often exceeding 60% overprediction in the core. In high-Re channel flows, models underpredicted near-wall peak turbulence energy whereas in low-Re channel flows, they showed better agreement near the wall but over-predicted values in the core. For the MHD flows, the implementation of the MHD sources improved predictions for low-Re k - ε models. The high-Re models which use the wall treatments did not show much improvement with MHD sources, perhaps due to the lack of MHD effects in the wall formulations.

In the case of low-Re square duct flows, the models tested did not predict the mean axial velocities to a good accuracy (error ranging ~8-30%) because of the secondary flows generated due to turbulence anisotropy. The TKE was overpredicted in the core, often exceeding ~60%, by all models except LB in MHD duct. The effect of turbulence suppression by magnetic field was not properly captured on mean velocity, Reynolds stresses/turbulent kinetic energy and frictional losses by any single model in a MHD duct, even after inclusion of the MHD sources of turbulence.

For problems involving high-Re, the SKE model offers reasonable accuracy at low computational cost. Adding EWT improves accuracy slightly over standard wall laws, but significantly increases cost. For flows with low-Re number, the Lam-Bremhorst (LB) low-Re k - ε model performed better than the others in both hydrodynamic and magnetic field influenced turbulent flows. Given the need to compute complex industrial flows with efficient computational use, using these 2 models with appropriate changes for magnetic field effects provides a reasonable compromise of accuracy and speed. Finally, the RSM-LPS model with EWT offers similar accuracy with the added ability of capturing turbulence anisotropy and secondary flows, but its computational cost is very high.

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